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Linear stability of flow in a differentially heated cavity via large-scale eigenvalue calculations

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Abstract Locates the onset of oscillatory instability in the fluid flow inside a differentially heated cavity with aspect ratio 2 by computing a steady-state and analyzing the stability of the system via eigenvalue approximation. Discusses the choice of parameters for the Cayley transformation so that the calculation of selected eigenvalues of the transformed system will reliably answer the question of stability. Also presents an argument that due to the symmetry of the problem, the first two unstable modes will have eigenvalues that are nearly identical, and the numerical experiments confirm this. Finally, locates a co-dimension 2 bifurcation signifying where there is a switch in the mode of initial instability. The results were obtained using a parallel finite element CFD code (MPSalsa) along with an Arnoldi-based eigensolver (ARPACK), a preconditioned Krylov method code for the necessary linear solves (Aztec), and a stability analysis library (LOCA).

1. Introduction

We locate the onset of oscillatory instability for the flow in a differentially heated cavity by computing a steady-state and analyzing its stability. We consider the flow in a box of width L = 1 and height H = 2 where the left and right vertical walls are held constant at different temperatures. Though this problem (with various values of Land H) has been the subject of much research (Janssen and Henkes, 1995; Le Quéré and Behnia, 1998; Mayne et al., 2000, 2001; Paolucci and Chenoweth, 1989; Xin and

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Linear stability

of flow

803

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Le Quéré, 1995; Xin *et al.*, 1997), most authors have predicted the onset of oscillatory convection by using transient calculations. An exception to this is the work of Xin and Le Quéré (2001), who have conducted a linear stability analysis in a square cavity using a direct method to solve the linear systems. While our work builds upon this body of knowledge, we differ in that we are using a general purpose finite element code and solving the resulting linear systems using iterative methods. This allows us to study complex geometries and solve much larger systems.

We have formulated the problem so that it is similar to the study by Paolucci and Chenoweth (1989). In this study they found that as the Rayleigh number is increased, boundary layers form on both vertical walls, and internal hydraulic jumps form at the corners. At a critical value of the Rayleigh number, the hydraulic jumps start oscillating. They also found that there were other frequencies of oscillation present in the system associated with "wall modes". The explanation in terms of hydraulic jumps was first proposed by Ivey (1984), but later authors (Ravi *et al.*, 1994) have objected to the hydraulic jump interpretation and instead refer to the oscillations of the hydraulic jumps as "internal wave instabilities."

In this paper, we verify these results by Paolucci and Chenoweth for the case of aspect ratio equal to 2 by matching frequencies of oscillations. We find an additional pair of modes with a lower frequency that destabilize the system at the lowest Rayleigh number. Our ability to calculate the stability of a steady-state solution allows us to draw some further conclusions not readily available with transient simulations. We observe numerically that the eigenvalues of this problem come in pairs where the eigenvalues are almost identical to each other. One of the eigenvalues in this pair is associated with a symmetric mode and the other with an anti-symmetric mode. We give a convincing analytical argument suggesting why this should be the case.

That the eigenvalues come in near identical pairs suggests that by varying other parameters (such as the Prandtl number and aspect ratio) in the problem, we can obtain one of these pairs to merge at precisely the point where the system loses its stability. This is an example of a double Hopf bifurcation, one of the five generic co-dimension 2 bifurcations (Guckenheimer and Holmes, 1983). The double Hopf bifurcation is especially interesting since it has a four dimensional center manifold, and we are almost guaranteed of getting chaotic behavior in the immediate vicinity of such a point in parameter space. By varying the Prandtl number in this problem we have been able to find a double Hopf bifurcation point. Additionally, the most interesting case we found resulted not from the eigenvalues in these symmetric/anti-symmetric pairs crossing, but from when an eigenvalue associated with a wall mode crossed an eigenvalue associated with an oscillating internal wave. This discovery was enabled by the complementary capabilities of calculating eigenvalues and eigenvectors and of tracking Hopf bifurcation points.

This classical problem not only exhibits interesting physical behavior, but also demonstrates and verifies our eigenvalue analysis capabilities. Our Cayley transform method, as implemented in the LOCA stability analysis library (Salinger *et al.*, 2002a), allows us to locate the onset of oscillatory instabilities; in order to locate these instabilities it is necessary to compute the eigenvalue of the system with largest real part (Meerbergen and Spence, 1997). It remains an open problem in large-scale non-symmetric eigenvalue calculations to reliably verify that the rightmost eigenvalue has been computed. Without that result, scientists and engineers interested in

804

HFF

14.6

computing linear stability require a variety of analysis tools; here we present a Cayley transform method that is effective in finding the rightmost eigenvalue when the imaginary part of that eigenvalue is large. One of the goals of this paper is to convince the reader of the reliability and applicability of this method to other problems of this type.

The flow due to natural convection in a differentially heated cavity is advectively dominated; advectively dominated flows are characterized by eigenvalues that have a large imaginary part relative to the real part. This can result in two computational difficulties. First, it can be difficult to compute the eigenvalues of the discretized system. Our choice of Cayley transform along with the use of an Arnoldi-based algorithm proves to be a reliable method to overcome this difficulty. The second difficulty is that we may need to discretize the Navier-Stokes equations on a highly resolved mesh so that the real part of the eigenvalues will approximate those of the continuous system.

The performance of our code on this problem demonstrates that we do need fine meshes to compute accurately converged real parts of the eigenvalues of interest. We will show that this is due to discretization errors, not to a failure of the eigensolver to compute the correct eigenvalues. In fact, we emphasize that the eigensolver handles with ease the large systems we are studying. Because the limitation lies in the discretization, we claim that we would have the same difficulty in accurately computing these flows using a transient finite element code.

Our calculations are carried out using a combination of a general purpose massively parallel unstructured grid finite element CFD code, MPSalsa (Shadid, 1999), and an existing Arnoldi-based eigensolver, ARPACK, (Lehoucq *et al.*, 1998) and a parallel iterative linear solver using preconditioned Krylov methods package, AZTEC (Tuminaro *et al.*, 1999). MPSalsa discretizes the Navier-Stokes equations and applies Newton's method to solve for the steady-state. Because our interest is in discretized Navier-Stokes equations in general geometries that lead to linear systems of order 10^4 - 10^7 for two- and three-dimensional problems, direct methods (even sparse direct methods) for the linear solves or subspace iteration for the eigensolve are not an option. We will demonstrate that parallel Krylov iterative methods can be reliably used for large-scale linear stability analysis on massively parallel machines.

Our approach is as reliable as calculations accomplished with transient methods; our approach is more efficient because we use a Krylov subspace method and use a frozen Jacobian, so we avoid the non-linear solve made at every time step by a transient calculation. While we cannot guarantee that our approach will reliably locate all instabilities because of the need to intelligently pick the parameters in the Cayley transformation, we assert that this is the same risk associated with choosing the time step and integration time when detecting instabilities through time integration. Moreover, our approach also provides qualitative information on the fluid flow not otherwise available. As we show, the information from the eigensolver can readily be used to track instabilities in parameter space and to locate higher co-dimension bifurcations.

We organize our paper as follows. In Section 2, we introduce our formulation of the problem of the flow in a differentially heated cavity that provides the numerical example for our study. We also state the Navier-Stokes equations with the Boussinesq approximation governing the motion of the flow and present a novel result regarding

Linear stability of flow

HFF	the symmetry of the problem and the resulting nearly identical eigenvalues. In Section
1/16	3, we discuss the finite element code MPSalsa, the Cayley transform as implemented in
14,0	the LOCA library, the choice of Cayley parameters and the Arnoldi-based eigenvalue
	package ARPACK. Section 4 gives linear stability analysis results for convection
	differentially heated cavity, including comparisons with published results and mesh
	resolution studies. In Section 5, we highlight some of the numerical issues that arise in
806	the linear stability analysis. Section 6 presents results of tracking instabilities as a
	function of the Prandtl number, including the detection of a co-dimension 2 bifurcation.
	Section 7 summarizes our findings.

2. Problem formulation

In this section, we describe the problem of convection in a two-dimensional vertical cavity and give the basic equations that govern our flow. We present the novel result that due to the symmetry of the problem we have a pair of nearly identical eigenvalues.

2.1 The problem of flow in a differentially heated cavity

We consider the flow in a cavity of width *L* and height *H*. The left vertical wall is held at a constant temperature $-\Delta T/2$, and the right vertical wall is held at the temperature $\Delta T/2$. We impose no-flux boundary conditions at the horizontal walls and no-slip boundary conditions on all walls.

We solve the Navier-Stokes equations with the Boussinesq approximation for the flow of a thermally driven incompressible fluid:

$$\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} + \frac{1}{\rho} \nabla p = \nu \nabla^2 \mathbf{u} + g \beta (T - T_{\text{ref}}) \mathbf{e}_{\text{g}}$$
(1)

$$\frac{\partial T}{\partial t} + \mathbf{u} \cdot \nabla T = \kappa \nabla^2 T \tag{2}$$

$$\nabla \cdot \mathbf{u} = 0 \tag{3}$$

where $\mathbf{u} = u\mathbf{e}_x + v\mathbf{e}_y + w\mathbf{e}_z$, p and T are the velocity, pressure and temperature; ρ , ν and κ are the density, kinematic viscosity and thermal diffusivity; g and β are the acceleration of gravity and the thermal expansion coefficient of the fluid. The vector \mathbf{e}_g is a unit vector in the direction of the gravity vector. The Boussinesq approximation assumes that the temperatures T are all close enough to an average temperature T_{ref} that we can ignore the variations in density in all terms in the equations except for the forcing term due to gravity. In these equations we subtract the hydrostatic part of the pressure.

The boundary conditions are zero velocities on all four walls, adiabatic Neumann conditions on the top and bottom walls for the heat equations, and Dirichlet temperatures on the side walls:

$$T\left(-\frac{L}{2},y\right) = \frac{\Delta T}{2}$$
 and $T\left(\frac{L}{2},y\right) = -\frac{\Delta T}{2}$.

Other than the physical constants appearing in the equations, the only parameters appearing in our problem are the temperature difference ΔT , the characteristic

geometrical length *L*, and the geometrical aspect ratio. The dimensionless parameters Linear stability that result from the parameters are the Rayleigh number, of flow

$$\mathrm{Ra} = \frac{g\beta\Delta TL^3}{\kappa\nu},$$

and the Prandtl number,

$$\Pr = \frac{\nu}{\kappa}$$

We achieve the desired Rayleigh and Prandtl numbers by selecting $\rho = L = \Delta T = 1$, $g = \Pr \times 10^{1}$, $\nu = \Pr \times 10^{-3}$ and $\kappa = 1 \times 10^{-3}$. We then control the Rayleigh number using Ra = $\beta \times 10^{7}$.

2.2 Symmetry and near-degeneracy of the eigenvalues

Because the right vertical wall is held at a temperature that is the negative of the left vertical wall, the governing equations are invariant under the following symmetry transformations:

$$Rz(\mathbf{x}) = \begin{pmatrix} -T(-\mathbf{x}) \\ -\mathbf{u}(-\mathbf{x}) \\ p(-\mathbf{x}) \end{pmatrix}$$
(4)

where we are representing our solution in the form

$$z(\mathbf{x}) = \begin{pmatrix} T(\mathbf{x}) \\ \mathbf{u}(\mathbf{x}) \\ p(\mathbf{x}) \end{pmatrix}$$

If $z(\mathbf{x})$ is a solution to our equations, then so is $Rz(\mathbf{x})$. However, it is not necessary that solutions to our equations satisfy $Rz(\mathbf{x}) = z(\mathbf{x})$.

We are analyzing the stability of symmetric solutions, so all eigenfunctions will either be symmetric or anti-symmetric. Any simple eigenfunction will either satisfy $R\phi(\mathbf{x}) = \phi(\mathbf{x})$ or $R\phi(\mathbf{x}) = -\phi(\mathbf{x})$. Symmetry can only be broken through a bifurcation, so that a solution that is initially symmetric will stay symmetric as we vary a parameter unless we encounter a bifurcation point.

When our system goes unstable, the internal waves will either oscillate in a symmetric manner or in an anti-symmetric manner. Physically we expect that if the walls are well separated, then the fluid on the left should be able to oscillate independently of the fluid on the right. In order for this to be so, we would have to be able to construct eigenfunctions where the fluid on the left oscillates but that on the right does not. The only way to do this is we should have multiple eigenvalues, with one symmetric eigenvector and the other anti-symmetric. This is not quite what occurs because the two sides are not completely separated, but we almost get this. Hence we have two eigenvalues that are almost identical to each other. This result is borne out in our eigenvalue calculations, presented in Section 4.

807

HFF 3. Methodology

In this section, we discuss the numerical methods used by MPSalsa to locate steady-state solutions of equations (1)-(3), the formulation of the eigenvalue problem and our Cayley transform method, and the numerical solution of the eigenvalue problem.

3.1 Spatial discretization and the non-linear solve

A full description of the numerical methods used by MPSalsa to locate steady-state solutions of equations (1)-(3) is available in the work of Shadid (1999) and the references are listed therein. A brief overview is presented in this section.

A mesh of quadrilaterals for two-dimensional problems and hexahedra for three-dimensional problems is generated to cover the domain. Although the code is written for general unstructured meshes to represent systems with complex geometries (Salinger *et al.*, 1999), the meshes used for the problem in this paper are structured. For parallel runs, the mesh is partitioned using the Chaco code (Hendrickson and Leland, 1995) in a way that will distribute work evenly while minimizing communication costs between processors.

A pressure stabilized Petrov Galerkin finite element method (PSPG-FEM) (Hughes *et al.*, 1986) is used to discretize the time-invariant versions of the governing partial differential equations (1)-(3) into a set of nonlinear algebraic equations. This formulation includes a pressure stabilization term so that the velocity components, temperature and pressure fields can all be represented with equal order nodal basis functions. This formulation does not include the upwinding terms that appear in the streamline upwind Petrov-Galerkin (SUPG) (Brooks and Hughes, 1982) and Galerkin/least-squares (GLS) formulations (Hughes *et al.*, 1989), because this term was found in a related study to increase the discretization error in computing bifurcation point over PSPG-FEM (Salinger *et al.*, 2002b). The PSPG-FEM method is a consistent stabilized scheme (as the SUPG and GLS schemes) because when the exact solution is inserted, the Boussinesq equations are satisfied exactly. We use bilinear and trilinear nodal elements for two- and three-dimensional problems, respectively.

Discretization of (1)-(3) results in the matrix equation

$$\begin{pmatrix} \mathbf{M} & 0 \\ \mathbf{N} & 0 \end{pmatrix} \begin{bmatrix} \dot{\mathbf{u}} \\ \dot{\mathbf{p}} \end{bmatrix} + \begin{pmatrix} \mathbf{K}_{u,T} + \mathbf{C}(\mathbf{u}) & -\mathbf{D} \\ \mathbf{D}^{\mathrm{T}} + \mathbf{G} & \mathbf{K}_{\mathrm{p}} \end{pmatrix} \begin{bmatrix} \mathbf{u} \\ \mathbf{p} \end{bmatrix} - \begin{bmatrix} \mathbf{g} \\ \mathbf{h} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$
(5)

where **u** is the vector of fluid velocity components and temperature unknowns, **p** the pressure, **M** the symmetric positive definite matrix of the overlaps of the finite element basis functions, $\mathbf{K}_{u,T}$ the stiffness matrix associated with velocity and temperature, $\mathbf{C}(\mathbf{u})$ the nonlinear convection, **D** the discrete (weak) gradient, \mathbf{D}^{T} the discrete (weak) divergence operator and \mathbf{K}_{p} the stiffness matrix for the pressure. **G**, \mathbf{K}_{p} , and **N** are stabilization terms arising from the PSPG-FEM. The vectors **g** and **h** denote terms due to boundary conditions and the Boussinesq approximation.

The resulting nonlinear algebraic equations arising from setting the time derivative terms to zero are solved using a fully coupled Newton-Raphson method (Shadid *et al.*, 1997). An analytic Jacobian matrix for the entire system is calculated and stored in a sparse matrix storage format. At each Newton-Raphson iteration, the linear system is

14,6

Linear stability solved using the Aztec package (Tuminaro et al., 1999) of parallel preconditioned Krylov iterative solvers. The accuracy of the steady-state solve is set by the following stopping criterion,

$$\left(\frac{1}{N}\sum_{i=1}^{N}\left(\frac{|\delta_i|}{\varepsilon_{\mathrm{R}}|x_i|+\varepsilon_{\mathrm{A}}}\right)^2\right)^{\frac{1}{2}} < 1.0,\tag{6}$$

where $\varepsilon_{\rm R}$ and $\varepsilon_{\rm A}$ are the relative and absolute tolerances desired, δ_i the update for the unknown x_i and N the total number of unknowns. We use relative and absolute tolerances of 10^{-5} and 10^{-8} , respectively, for this study. In Aztec we exclusively use an unrestarted GMRES iteration with a non-overlapping Schwarz preconditioner where an ILU preconditioner is used on each subdomain (each processor contains one subdomain). These methods enable rapid convergence to both stable and unstable steady-state solutions. The scalability of these methods to large system sizes and number of processors is demonstrated by the solution of a 16 million unknown model on 2,048 processors (Burroughs et al., 2001).

3.2 The discretized eigenvalue problem and Cayley transforms

The PSPG-FEM results in a spatial discretization of the Navier-Stokes equations with the Boussinesq approximation. This leads to a finite dimensional system of differential algebraic equations of the form

$$\mathbf{B}\dot{\mathbf{x}} = \mathbf{F}(\mathbf{x}), \quad \mathbf{x}(0) = \mathbf{x}_0, \tag{7}$$

where the matrix \mathbf{B} is singular (due to the divergence free constraint) and \mathbf{x} is a vector containing the nodal values of the velocities, temperature and pressure at the nodes of the finite element mesh. Because of the stabilization terms in the PSPG discretization, **B**, the matrix associated with the time derivative term in equation (5), is a non-symmetric matrix.

One can determine the stability of a steady-state solution \mathbf{x}_s of $\mathbf{F}(\mathbf{x}_s) = 0$ in one of the two ways: by solving the generalized eigenvalue problem that results from the linearization of equation (7) about the steady-state, or by using a time integration scheme.

The first approach solves the generalized eigenvalue problem

$$\lambda \mathbf{B} \mathbf{z} = \mathbf{J} \mathbf{z}.$$
 (8)

that arises from the linearization of equation (7) about the steady-state. The matrix **J** is the Jacobian of $F(\cdot)$ linearized about \mathbf{x}_s . We assume that the eigenvalues are ordered with respect to decreasing real part; real $(\lambda_{i+1}) \leq \operatorname{real}(\lambda_i)$. If all the eigenvalues of problem (8) have negative real parts, then the steady-state is stable.

We use a Cayley transform so that we find the eigenvalues γ_i of the system

$$(\mathbf{J} - \sigma \mathbf{B})^{-1}(\mathbf{J} - \mu \mathbf{B})\mathbf{z} = \gamma \mathbf{z}$$

that are related to the eigenvalues λ_k of problem (8) via

$$\gamma_i = \frac{\lambda_k - \mu}{\lambda_k - \sigma}$$
 $i = 1, ..., n;$ $k = 1, ..., n$

809

of flow

HFF 14,6 We choose $\sigma > 0$ and $\mu = -\sigma$; we choose the value of σ so that it is of similar magnitude to the imaginary part of the eigenvalue of interest, and so $\sigma > \operatorname{Re}(\lambda_1)$. This transformation has the property of mapping a λ in the right half of the complex plane (i.e. an unstable mode) to a γ outside the unit circle, and those on the left half plane (i.e. a stable mode) to a γ inside the unit circle. That is,

810

 $\operatorname{real}(\lambda) > 0 \Rightarrow \|\gamma\| > 1.0$ and $\operatorname{real}(\lambda) < 0 \Rightarrow \|\gamma\| < 1.0$.

Since Arnoldi's method will converge more rapidly to those eigenvalues with larger magnitudes, this is a very desirable property for calculating eigenvalues for use in linear stability analysis.

The use of preconditioned Krylov methods for both eigenvalue problem and ensuing linear solves for large-scale two- and three-dimensional problems is not generally undertaken. The results of our paper will show that we have found success in this method. The computation of eigenvalues of the linearized steady-state has received much attention in the last 15 years (Christodoulou and Scriven, 1988; Cliffe et al. 1993; van Dorsselaer, 1997; Edwards et al., 1994; Fortin et al., 1997; Lehoucg and Salinger, 2001; Mittelmann et al., 1994; Morzyńki et al., 1999; Tukerman et al., 2000). The consensus of this research is to convert the generalized eigenvalue problem (8) into a standard eigenvalue problem and then solve the resulting set of linear equations during each iteration of the eigensolver. Most of the authors of these papers then solve the eigenvalue problem using inverse subspace iteration or Arnoldi's method with a sparse direct method for the resulting linear set of equations (Christodoulou and Scriven, 1988; Cliffe et al., 1993; van Dorsselaer, 1997; Fortin et al., 1997; Morzyńki et al., 1999; Mittelmann *et al.*, 1994). This typically limits the linear stability analysis to two-dimensional problems. Our approach of using Cayley transforms to reduce problem (8) to a standard eigenvalue problem is successful, and the eigensolver performs with ease on our large (order 10^{5} - 10^{7}) systems.

The second approach to computing the stability of a steady-state is to use a time integration scheme; standard time integration schemes typically perform a nonlinear solve (due to convection) at every time step. We can think of these as computing an iteration of the form

$$\mathbf{x}^{n+1} = \mathbf{G}(\mathbf{x}^n). \tag{9}$$

The iteration is initialized with an iterate near the steady-state and if the iteration converges toward the fixed point \mathbf{x}_s , then the steady-state is declared stable. If \mathbf{x}_0 is an initial condition for equation (9), then the convergence and numerical stability of the fixed point iteration is determined by the spectral radius of the Jacobian of $\mathbf{G}(\cdot)$. In particular, denote the eigenvalues of $\mathbf{G}_{\mathbf{x}}(\mathbf{x}_0)$ by γ_i ordered so that $|\gamma_{i+1}| \leq |\gamma_i|$.

A popular time integration scheme is given by the trapezoidal rule and results in the iteration

$$\mathbf{x}^{n+1} = \mathbf{G}(\mathbf{x}^n) = \left(\mathbf{B} - \frac{\Delta t}{2}\mathbf{J}\right)^{-1} \left(\mathbf{B} + \frac{\Delta t}{2}\mathbf{J}\right) \mathbf{x}^n \tag{10}$$

where the Jacobian is "frozen" at the steady-state. The eigenvalues γ_i and λ_i are related via

$$\gamma_i = -\frac{\lambda_k + \frac{2}{\Delta t}}{\lambda_k - \frac{2}{\Delta t}} \quad i = 1, ..., n; \quad k = 1, ..., n$$
 Linear stability of flow

and so, in principle, the eigenvalues of problem (8) can be determined by computing those of

$$-(\mathbf{J} - \sigma \mathbf{B})^{-1}(\mathbf{J} - \mu \mathbf{B})\mathbf{z} \equiv \mathbf{G}\mathbf{z} = -\gamma \mathbf{z}$$

where $\mu = -\sigma = 2/\Delta t$. Note that this is the same as our choice of Cayley transform with $\mu = -\sigma = 2/\Delta t$.

The above discussion demonstrates that at a steady-state, time integration and computing the eigenvalues of problem (8) are intimately related when a frozen Jacobian approximation is employed. We remark that although large-scale eigensolvers (subspace iteration or Arnoldi's method) favor the computation of those eigenvalues largest in magnitude, these may not be the desired rightmost eigenvalues. This occurs when the flow is advectively dominated. Our choice of a Cayley transform allows us to overcome this difficulty.

We now explain why Arnoldi's method for the eigenvalue solvers is preferred to the typically undertaken transient calculation. A transient calculation (with the linearized Jacobian **J**) or fixed point iteration is equivalent to the power method on **G**. The rate of convergence to the eigenvector associated with γ_1 is $|\gamma_2/\gamma_1|$. The rate of convergence improves to $|\gamma_{m+1}/\gamma_1|$ if the power method is replaced by subspace iteration on *m* vectors. However, the resulting rate of convergence can be intolerable. The rate of convergence to $\gamma_1, \gamma_2, \ldots, \gamma_r$ may be dramatically improved by projecting **G** onto the column space of

$$\mathbf{x}^0, \mathbf{x}^1, \ldots, \mathbf{x}^m$$
.

Arnoldi's (1951) method iteratively determines an orthogonal basis for the above column space that by definition is a Krylov subspace.

3.3 Arnoldi's method and the numerical solution of the eigenvalue problem

The remainder of the section reviews several issues with the use of Arnoldi's method for the numerical solution of the eigenvalue problem. We use the parallel implementation P_ARPACK (Maschhoff and Sorensen, 1996) of ARPACK (Lehoucq *et al.*, 1998) for computing the eigenvalues of problem (8) via Cayley transforms. We refer the reader to see the work of Lehoucq and Salinger (2001) for information regarding the use of ARPACK for problems in linear stability analysis.

We discuss the selection of the Cayley parameters σ and μ . There are two strategies by which we can choose the Cayley parameters. The first strategy was presented in the previous subsection and draws upon a connection with the trapezoidal rule in fixed point iteration. This is the strategy we employ in this study; we will discuss the implications of this choice in Section 5. The second strategy was presented by Lehoucq and Salinger (2001); the Cayley parameters are selected $\lambda_1 < \sigma < \mu$ so that the condition number of $(\mathbf{J} - \sigma \mathbf{B})^{-1}(\mathbf{J} - \mu \mathbf{B})$ is bounded and so can be efficiently solved with preconditioned Krylov methods. This second strategy tends to be more efficient than the first strategy for finding eigenvalues with zero or small imaginary parts; however, it is not as reliable. (Nor is there a relationship with a common fixed point iteration scheme for determining the stability of the steady-state. The analogous time-stepper is unconditionally unstable for all modes.) The lack of reliability manifests itself when the flow is advectively dominated so that the rightmost λ s do not correspond to the largest in magnitude γ s. We remark that we encountered this unreliability in the solution of the problem of the secondary bifurcation from steady rolls into oscillatory rolls in the Rayleigh-Bénard problem, discussed by Burroughs *et al.* (2001): the first strategy finds the eigenvalues of interest while the second does not.

HFF

14,6

812

We briefly overview several salient issues. Further details are available in the discussion of the numerical experiments performed in Sections 4 and 5, and in the work of Lehoucq and Salinger (2001).

- (1) The numerical solution of the linear system resulting from using a Cayley transform is found by exclusively using an unrestarted GMRES iteration with a non-overlapping Schwarz preconditioner where an ILU preconditioner is used on each sub-domain (each processor contains one sub-domain).
- (2) We must choose the size of the Arnoldi space *m* (needed by ARPACK). Our findings, in general, are that for the most difficult problems *m* was never larger than 160 and 80 was typically more than adequate. We remark that although ARPACK does provide a capability to restart the Arnoldi iteration, our experiments did not use this capability. Instead, our focus is to examine the use of preconditioned Krylov methods for linear stability analysis.
- (3) The tolerance needed by the GMRES iteration and ARPACK and their relationship was studied by Lehoucq and Salinger (2001), and adjusts automatically to the scaling of the problem. In general, these tolerances were no larger than 10^{-6} and no smaller than 10^{-9} .
- (4) Since the Boussinesq equations (1)-(3) model an incompressible fluid, the starting vector for ARPACK is selected as $\mathbf{J}^{-1}\mathbf{B}\mathbf{w}$, where \mathbf{w} is a random vector. The resulting vector is divergence free (Meerbergen and Spence, 1997).
- (5) The P_ARPACK subroutines pdnaupd and pdnaupd were modified to implement the Cayley transform and an improved check for termination. The eigensolve is terminated when $\lambda_1, \lambda_2, \ldots, \lambda_r$ and corresponding approximate eigenvectors for a user specified r satisfy the residual tolerance. This code is available through the LOCA library (Salinger *et al.*, 2002a).

4. Results of convection in a differentially heated cavity

We conduct our numerical experiments at Pr = 0.71 and with H = 2 and L = 1 in order to compare our results with those of Paolucci and Chenoweth (1989). We validate our results through a comparison with the numerical solutions of Paolucci and Chenoweth and verify our results by tracking the residual accuracy of our computed eigenvalues and linear systems and through a study of convergence as we refine the finite element mesh.

Although Paolucci and Chenoweth did not make the Boussinesq approximation in their calculations, they purposely used conditions that are well approximated by the Boussinesq approximation. In particular, $\Delta T/T_{AV} = 0.01$ where ΔT is the difference

between the wall temperatures and T_{AV} is the average of the wall temperatures. When A = H/L = 2 they found a Rayleigh number of approximately 3×10^7 with a dimensionless frequency of f = 173.2. (Because we do not make our equations dimensionless in the same way, to compare the frequencies f_{PC} reported in Paolucci and Chenoweth to the imaginary part of our computed eigenvalues we look at $\omega = f_{PC} \times 2\phi/1,000.$)

We solve quadrilateral finite element meshes with bilinear basis functions of 40×80 , 80×160 , 160×320 , 320×640 and $640 \times 1,280$. The spacing between the mesh points increases exponentially as we move away from the walls, with the points in the middle of the box having mesh spacings about 20 times as large as those near the walls.

For the finest mesh, we have 3,284,484 unknowns and solve on 256 processors of the Sandia-Intel TFlop machine (ASCI Red) with 333 MHz Pentium processors. On this final mesh it is somewhat difficult to achieve convergence to the steady-state solution; we rely on continuation to find the steady-state at the desired Rayleigh numbers. The number of GMRES iterations for each eigensolver iteration is approximately 200. The time to compute eigenvalues for the finest mesh is 6 h for Ra = 3.0×10^7 . We set the Cayley parameters $\sigma = 5$, $\mu = -5$ and the Arnoldi size to 160.

Table I shows the eigenvalues for the 160×320 mesh and how they compare with the results of Paolucci and Chenoweth. Paolucci and Chenoweth performed calculations at Rayleigh numbers of 3×10^7 and 2×10^7 for A = 2.0. The frequency they report at Ra = 3×10^7 is in excellent agreement with the frequency predicted by our eigenvalue calculation (1.088 vs 1.097). When Ra = 2×10^7 we still get good agreement (2.316 vs 2.338), but the frequency they report agrees with what we calculate to be the third mode. While they report the flow as being stable, our eigenvalue calculations report that the flow is unstable because the first two modes have positive real parts. Possible explanations for why the previous work may have missed this mode include that the ungraded mesh used for this data point (generated with the computing power available 14 years back) may not have fully resolved the flow or that the starting point for the transient calculation did not contain a significant contribution in the direction of this instability (which is very close to being neutrally stable).

In order to see how the steady-state solution converges with mesh refinement we have included Table II. This table shows the three most unstable eigenvalues and the maximum value of the *x*-velocity calculated with our various meshes. We are clearly getting convergence, but the convergence of the maximum *x*-velocity with mesh is

Ra(10 ⁷)	ω_1	ω_2	λ_1	λ_2	λ_3
3.0 2.75 2.5 2.25 2.0 1.75	1.088	2.316	$\begin{array}{c} 0.3295 \pm 1.097 \mathrm{i} \\ 0.2678 \pm 1.056 \mathrm{i} \\ 0.1937 \pm 1.010 \mathrm{i} \\ 0.1067 \pm 0.9584 \mathrm{i} \\ 0.0138 \pm 0.8946 \mathrm{i} \\ -0.0631 \pm 0.8143 \mathrm{i} \end{array}$	$\begin{array}{c} 0.3259 \pm 1.099i \\ 0.2634 \pm 1.0574i \\ 0.1884 \pm 1.012i \\ 0.1001 \pm 0.9628i \\ 0.0001 \pm 0.9081i \\ - 0.0649 \end{array}$	$\begin{array}{c} 0.0961 \pm 12.14 \mathrm{i} \\ 0.0474 \pm 11.44 \mathrm{i} \\ - 0.0017 \pm 10.73 \mathrm{i} \\ - 0.0479 \pm 9.997 \mathrm{i} \\ - 0.0681 \pm 2.338 \mathrm{i} \\ - 0.0757 \pm 2.177 \mathrm{i} \end{array}$

Note: ω_1 and ω_2 are based on the frequencies reported by Paolucci and Chenoweth (1989) and are available for comparison for the two Rayleigh numbers 3.0×10^7 and 2.0×10^7

Linear stability of flow

Table I.

The eigenvalues for convection in a cavity with mesh 160×320 somewhat slow and clearly is no better than the convergence with mesh of the eigenvalues.

We see slow convergence toward the real parts of the most unstable eigenvalue. (Other test problems we have studied that are not strongly advectively dominated flows show quadratic convergence rates (Burroughs *et al.*, 2001).) We believe that this problem demonstrates the limitations of looking for grid independence with a linear basis functions, particularly for highly advective flows. However, we note that the difficulties are with the resolution of the discretization and not in solving the eigenvalue problem. We emphasize that a transient solution is not any more reliable than the eigenvalue computations, and that in fact our eigensolver encounters no trouble in this 3 million unknown system. We also note that this problem is two-dimensional; if we were trying to achieve the same resolution on a three-dimensional problem, we would have billions of unknowns.

5. Numerical issues

Because we use parallel preconditioned Krylov iterative methods for the eigenvalue problem and resulting linear sets of equations, our results are obtained by specifying the values of certain adjustable parameters: we need to specify the Cayley parameters σ and μ and the size of the Arnoldi space. We briefly review our verification procedures used for our numerical experiments; as several issues have been discussed in an earlier paper, which used the same CFD Code, MPSalsa, and eigensolver, P_ARPACK, but a different Cayley method, the reader is referred to Lehoucq and Salinger (2001) for information regarding details of linear algebra tolerances. Our main emphasis in this section is to illustrate how sensitive our results are to the Cayley parameters.

Denote by λ_c and \mathbf{z}_c the approximations to an eigenvalue and eigenvector of problem (8). We verify these approximations by computing the norm of the residual

Direct residual =
$$\frac{\|\mathbf{J}\mathbf{z}_c - \lambda_c \mathbf{B}\mathbf{z}_c\|}{\|\mathbf{B}\mathbf{z}_c\|},$$
 (11)

where $\|\cdot\|$ is the Euclidean norm of a vector. These errors vanish only when λ_c and \mathbf{z}_c are an eigenpair for problem (8). Note that these measures are independent of the scaling of \mathbf{z}_c .

We now discuss the Cayley parameters and the size m of the Arnoldi space used by ARPACK. These two parameters are related because if one chooses the Cayley parameters poorly, a large Arnoldi space will be required to obtain accurate

	Ν	λ_1	λ_2	λ_3	x-velocity	coordinates
	40	$0.3217 \pm 1.020i$	$0.3192 \pm 1.020i$	$-0.0778 \pm 2.856i$	0.8054	(0.119, 1.97)
	80	$0.3326 \pm 1.091i$	$0.3294 \pm 1.092i$	$-0.0003 \pm 11.98i$	0.7993	(0.129, 1.97)
	160	$0.3295 \pm 1.097i$	$0.3295 \pm 1.099i$	$0.0961 \pm 12.14i$	0.8032	(0.124, 1.97)
	320	$0.3275 \pm 1.096i$	$0.3238 \pm 1.098i$	$0.1040 \pm 12.19i$	0.8048	(0.124, 1.97)
	640	$0.3267 \pm 1.096i$	$0.3231 \pm 1.097i$	$0.1039 \pm 12.20i$	0.8052	(0.122, 1.97)
uted	Note: with y	For the velocity in th varving mesh resoluti	e problem of the onse on of $N \times 2N$	t of convection in a hea	ted cavity with	$Ra = 3.0 \times 10^7$

Table II. Eigenvalues and maximum comput values

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814

eigenvalues. Our experience dictates that it is best to choose the Cayley parameters so that they are of the same order of the imaginary part of the most unstable eigenvalue. We believe that this is a reasonable assumption because the user typically has some idea of the location of the imaginary part of the most unstable eigenvalue. For example, this information is available if we are solving a problem that is a small variation of a problem that has already been solved or if we have access to related experimental or computational results. This is a drawback to the method if there is no prior evidence regarding the size of the imaginary portion of the most unstable eigenvalue. However, – this is the same issue as choosing a time step size for transient runs that is not so large as to step over oscillations, or a total time that is too small to sense the oscillations.

Table III shows the errors in the most unstable eigenvalue of the onset of convection in a differentially heated cavity as a function of the Cayley parameters and the size of the Arnoldi space. These calculations were accomplished with a 160×320 mesh and a Rayleigh number of 3.0×10^7 . We see that changing the Cayley parameters from ± 1 to ± 0.5 or ± 5 does not significantly degrade the performance of the algorithm. By the time the Cayley parameters are ± 20 we are seeing some degradation in the algorithm, but we are still getting quite good convergence after 160 iterations. Choosing the Cayley parameters too large is the same as integrating in time with too small a time step; it requires more Arnoldi iterations (time steps) to detect an oscillation. Notice that we also sometimes misidentify the most unstable eigenvalue; looking at the error, though, we see that this misidentified eigenvalue has not converged to a reasonable

$\sigma = -\mu$	Arnoldi size	Eigenvalue	Direct residual	
0.5				
0.0	40	$0.3295 \pm 1.097i$	5.063×10^{-8}	
	80	$0.3295 \pm 1.097i$	4564×10^{-8}	
	160	$0.3295 \pm 1.097i$	4564×10^{-8}	
1	100	0.0200 = 1.0011	1.001 × 10	
1	40	$0.3295 \pm 1.097i$	2597×10^{-8}	
	80	0.0230 = 1.0071 0.4774 + 13.03i	8.685×10^{0}	
	00	$0.3295 \pm 1.097i$	2904×10^{-8}	
	160	0.3233 = 1.0371 0.8677 + 17.39i	7.885×10^{0}	
	100	$0.3295 \pm 1.097i$	2904×10^{-8}	
	200	0.3295 ± 1.0971	2.004×10^{-8}	
5	200	0.5255 ± 1.0571	2.504 × 10	
5	40	$0.5761 \pm 12.08i$	3726×10^{-1}	
	40	0.3701 ± 12.301 $0.3205 \pm 1.007i$	5.720×10^{-5}	
	20	0.3293 ± 1.0971 0.2205 ± 1.0073	7.709×10^{-7}	
	00 100	0.3293 ± 1.0971	2.440×10 7.001 × 10 ⁻⁸	
00	160	0.3295 ± 1.0971	7.281 × 10	
20	10	0.0004 + 17.05	0.05010-1	
	40	0.6094 ± 17.351	3.056×10^{-1}	
		$0.4343 \pm 20.11i$	4.553×10^{-1}	
		$0.2769 \pm 1.060i$	4.801×10^{-2} Th	1e effe
	80	$0.3272 \pm 1.096i$	2.256×10^{-4}	and (
	160	$0.3300 \pm 1.098i$	8.196×10^{-5}	

Note: These results are for the most unstable eigenvalue at $Ra = 3.0 \times 10^7$ and a grid of 160×320 . In the case where the eigenvalue of interest, 0.3295 ± 1.097 i, is not identified as the most unstable eigenvalue, we have listed both eigenvalues

Linear stability

of flow

815

e effect of Arnoldi size and Cayley parameters on the problem of convection in a differentially heated cavity tolerance. In these situations increasing the size of the Arnoldi space allows us to compute the eigenvalues more accurately.

The accuracy of all of these calculations can also be limited by the accuracy to which we solve our linear systems at each Arnoldi iteration. For example, in Table III we do not get appreciably better results by using an Arnoldi space of size 160 instead of 80. In general, to improve the accuracy of our eigenvalue calculations we must either increase the size of the Arnoldi space or choose a better value for $\mu = -\sigma$ or decrease the tolerance to which we solve our linear systems. We should note, however, that these eigenvalue calculations are already highly converged. Even those with residuals near 10^{-4} instead of below 10^{-7} had the eigenvalues correct to three digits. A comparison to the mesh convergence results in Table II indicates that the limiting factor in predicting the eigenvalues to the real PDE system is more likely to be the discretization than the eigensolver.

6. Combining linear stability with bifurcation tracking

The method of determining solution stability through eigenvalue calculations of steady solutions lends itself to the use of bifurcation tracking algorithms. In particular, we use the Hopf bifurcation tracking algorithm that was presented and verified by Salinger *et al.* (2002b) for the benchmark problem of flow in a differentially heated cavity of aspect ratio 8. In this section, we show how the combined capabilities of linear stability and bifurcation tracking can be used to provide considerable insight into the stability picture for the model problem of flow in a differentially heated cavity of aspect ratio 2.

The results in Table I indicate that, with the 160×320 mesh, the first instability for a fluid with Pr = 0.71 occurs for $1.75 \times 10^7 < \text{Ra} < 2.0 \times 10^7$. Using the solution vector, eigenvector, and imaginary part of the eigenvalue at Ra = 2.0×10^7 , we invoked the Hopf bifurcation tracking algorithm in LOCA (Salinger *et al.*, 2002a). This algorithm uses a Newton algorithm to directly solve the Hopf bifurcation and requires a good initial guess as supplied by the eigensolver.

The Hopf tracking algorithm located the first instability, which we will term IA, at $Ra = 1.9608 \times 10^7$ and the second, IS, at $Ra = 1.9997 \times 10^7$. Visualization of the eigenmodes shows that the first mode is anti-symmetric with respect to the symmetry of the equations, as shown in equation (4), while the second is the symmetric version of the same physical mode.

We became curious about how persistent was the phenomenon that the anti-symmetric mode is the first to lose stability as a function of another system parameter. We tracked the Rayleigh number where the Hopf bifurcation occurs as a function of the Prandtl number. We did not find a change in the order of instability as we increased to Pr = 1.3. However, when decreasing the Prandtl number to generate the IA and IS curves in Figure 1, we found that the curves cross at Pr = 0.6368 and $Ra = 2.908 \times 10^7$, indicating that indeed the symmetric mode becomes more unstable than the anti-symmetric mode for Prandtl numbers in the neighborhood below that.

However, verification of this double-Hopf bifurcation with the eigensolver led to the discovery of two other complex pairs of eigenvalues with positive real parts. Further computations produced the curves labeled WA and WS in Figure 1. These modes are the anti-symmetric and symmetric versions of the wall mode described by Paolucci and Chenoweth (1989). We can see graphically that a co-dimension 2 bifurcation occurs

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817

Figure 1. The tracking of four Hopf bifurcations as a function of the Prandtl number shows that the IA mode goes unstable at the lowest Ra until Pr = 0.681, at which time the WA mode is the first to go unstable. These modes are shown in Figure 3

Note: The dotted line extensions to the **WA** and **WS** branches were added to clarify that these branches will continue to lower values of Pr, but these parts have not been calculated

near Pr = 0.681 and $Ra = 2.22 \times 10^7$. At this Prandtl number there is a switch between whether the IA or WA mode is the first to go unstable. Figure 2 shows a visualization of the base flow and temperature contours at this point, and Figure 3 shows the temperature profiles for both modes that go unstable at this point. Since these are oscillatory instabilities, both real and imaginary parts of the eigenvectors are visualized for each mode.

As the two wall modes continue to lower Prandtl numbers, they also appear to cross. At this point, convergence was lost for the anti-symmetric mode. One interesting point is that this crossing of branches WA and WS occurs where the frequencies appear to be equal, while this was not the case when the IA and IS modes cross. This added degeneracy could be responsible for the difficulties in convergence.

7. Conclusions

We have completed a linear stability analysis on the problem of the flow in a differentially heated cavity. We have identified the frequency of the oscillatory instability for various Rayleigh numbers for an aspect ratio of 2 and a Prandtl number of 0.71. The frequency we identify at $Ra = 3.0 \times 10^7$ is in excellent agreement with the prior published results, but for $Ra = 2.0 \times 10^7$ we find two modes more unstable than that found by Paolucci and Chenoweth (1989), and the frequency of the third most unstable mode corresponds to their published result. We also present an argument that the first two most unstable modes will have eigenvalues that are nearly identical, and our eigenvalue calculations demonstrate this is the case.

We have demonstrated both capabilities and limitations of using a general purpose finite element code and eigensolver for fluid stability calculations. Our interest is in large problems in possibly complex geometries where it is necessary to use iterative methods for the linear algebraic calculations. Our method has proved to be reliable in HFF 14,6

818

Figure 2.

This plot shows the steady solution where it loses stability with respect to two modes, at the co-dimension 2 bifurcation where the IA and WA branches cross in Figure 1



Note: Velocity vectors and temperature contours are shown for this symmetric solution

identifying the most unstable eigenvalue in advectively dominated flows because of our choice of Cayley transforms employed. The limitation of our method is that it is computationally intensive to reach high levels convergence with a low order finite element discretization. We do not believe that our eigenvalue techniques have reached any inherent limitation.

In flows that are advectively dominated, computing stability using either an eigensolver or transient calculations will produce the same difficulties in that they will require a fine discretization of the finite element mesh. We maintain that our results are as reliable as those obtained using transient integration, but that our results are more efficiently computed because we use a Krylov subspace method instead of the power method, and because we use a frozen Jacobian. We believe that our use of preconditioned Krylov iterative methods were successful because of the high quality and robust implementation of these algorithms, ARPACK and Aztec.

Determination of stability through calculation of steady-states and leading eigenvalues and eigenvectors lends itself well to using bifurcation tracking algorithms. We have shown the power of using these complementary techniques by uncovering the stability behavior for a range of Prandtl number. A co-dimension 2 bifurcation representing the exchange of initial instability between interior and wall modes was found to exist with just a 5 percent decrease in the Prandtl number from the conditions studied earlier.



Linear stability of flow

819

Note: Temperature contours for the real and imaginary components of the anti-symmetric interior mode **IA** are shown on top, and those for the anti-symmetric wall mode **WA** are shown on the bottom

Figure 3. The modes of instability at the co-dimension 2 bifurcation are visualized

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821

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